



LUND
UNIVERSITY

Methanol Engine Research at Lund University

2016-06-15

Martin Tunér

Division of Combustion Engines
Department of Energy Sciences
Lund University

GreenPilot kick off seminar, 15-16th of June in Gothenburg

Outline

- Engine Research at Lund University
- Benchmarking of Alternative Fuels and Engine Concepts
- Theoretical Emissions Potential for Methanol Combustion
- Methanol Engine Combustion Concepts Under Investigation
- Conclusions and Research Opportunities



Lund University



- Founded in 1666
- 46 000 students
- 2 800 doctoral students
- 6 000 employees
- 562 professors (18 % women)
- Eight faculties
- Several campuses
- EUR 750 million turnover
- 2/3 Research – 1/3 Education





LUND UNIVERSITY

Lund University
46000 students

Faculty of Engineering
(Lund Institute of Technology)
9600 students

Department of Energy Sciences
(former Heat and Power Engineering)
(Staff 90 persons)

Department of Engineering Physics

Department of Automatic Control

Prof. Anders Rantzer
(staff 35 persons)

Division of Energy economics and planning
Prof. J. Pyrko

Division of Thermal Power Engineering
Prof. Jens Klingmann

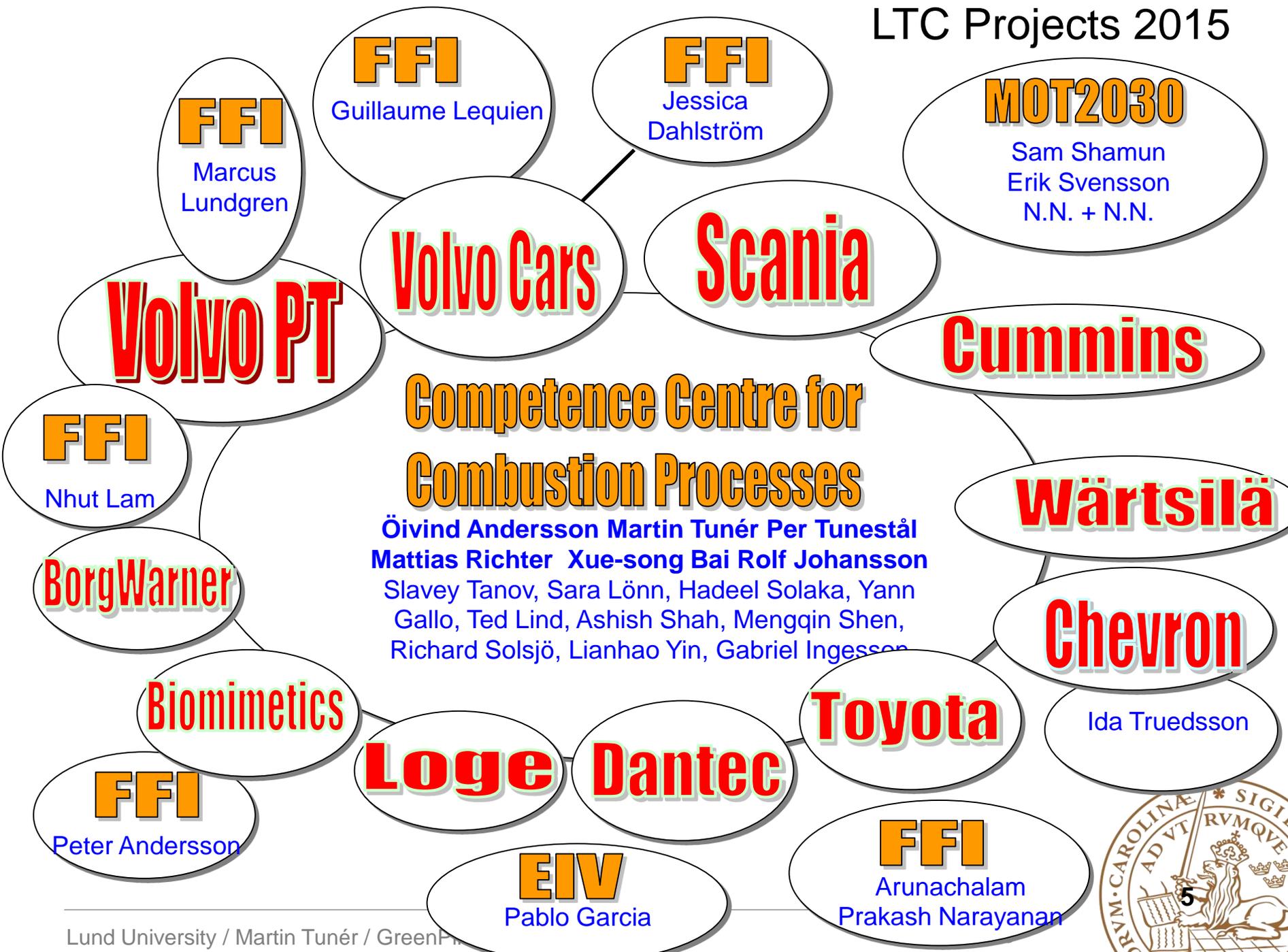
Division of Heat transfer
Prof. Bengt Sundén

Division of Fluid dynamics
Prof. Johan Revstedt

Division of Combustion Engines
Prof. Öivind Andersson
(staff 32 persons)

Division of Combustion Physics
Prof. Marcus Aldén
(staff 40 persons)





Lund Engine Labs

Light duty engines

1. Volvo D5 optical engine for PPC
2. Volvo D5 optical engine for diesel
3. Volvo VEA multi cylinder for PPC
4. Volvo VEA multi cylinder for LTC
5. CFR engine for fuel testing
6. Volvo VEP for education/projects

Heavy duty engines

1. Scania D13 optical engine for PPC
2. Scania D13 optical engine for diesel
3. Volvo D13 optical engine for PPC
4. Scania D13 single cylinder engine for PPC
5. Volvo D13 six-cylinder for PPC
6. Volvo D13 six-cylinder for dual fuel/HAM
7. Scania D13 six-cylinder for PPC

Medium speed engine

1. Wärtsilä 20 optical engine

– KCFP



MOT-2030

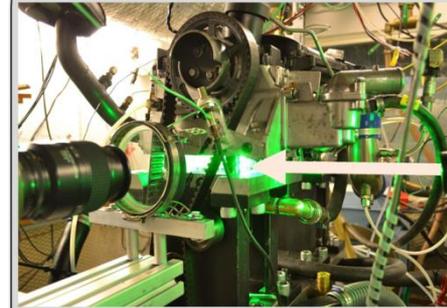
Highly Efficient Methanol Engine Systems
for Fossil Free Transportation 2030

Engine experiments



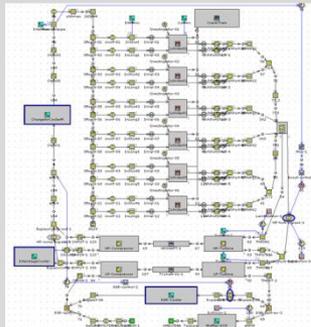
Investigate/explain emissions and efficiency performance of methanol in PPC engines

Optical diagnostics



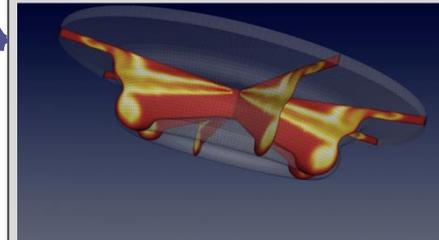
Investigate/explain various phenomenon of methanol combustion inside the PPC engine

System analysis



Investigate/explain overall Well-To-Wheel PPC engine performance with methanol

Advanced modeling



Investigate/explain relations between chemistry, heat release, heat transfer and engine physical parameters for methanol PPC

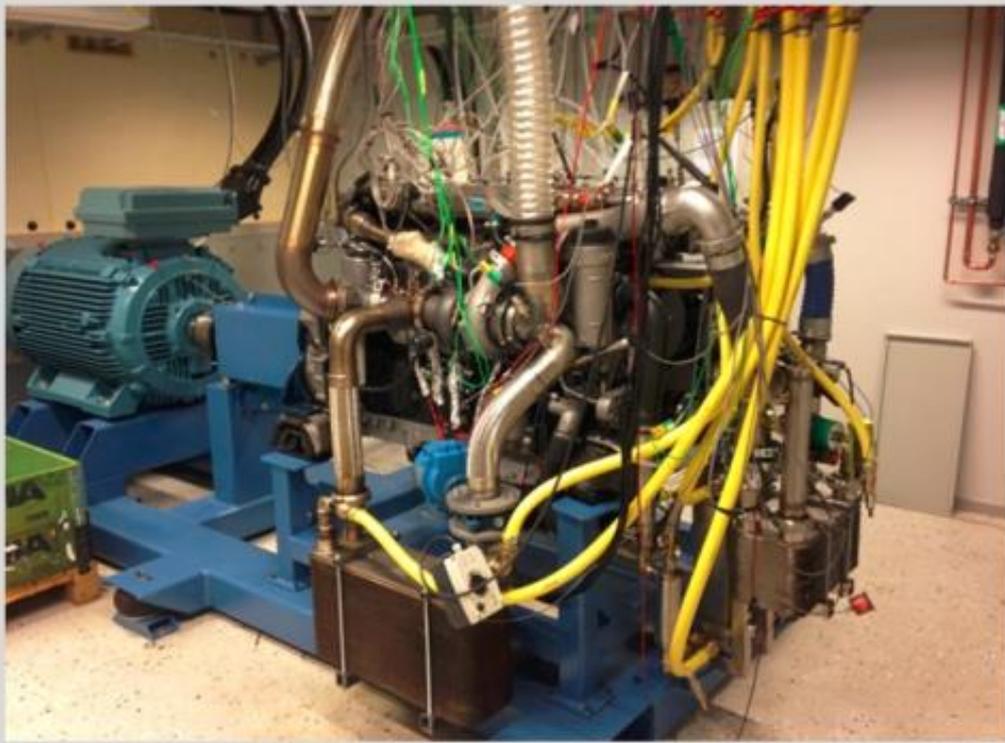
Supporting companies



In-house tests and other experience of methanol fuel in engines

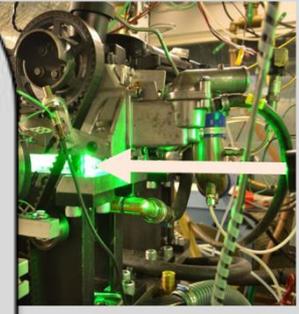


Engine experiments



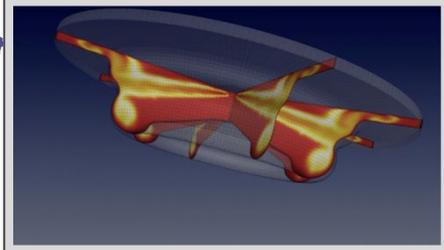
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MOT-2030

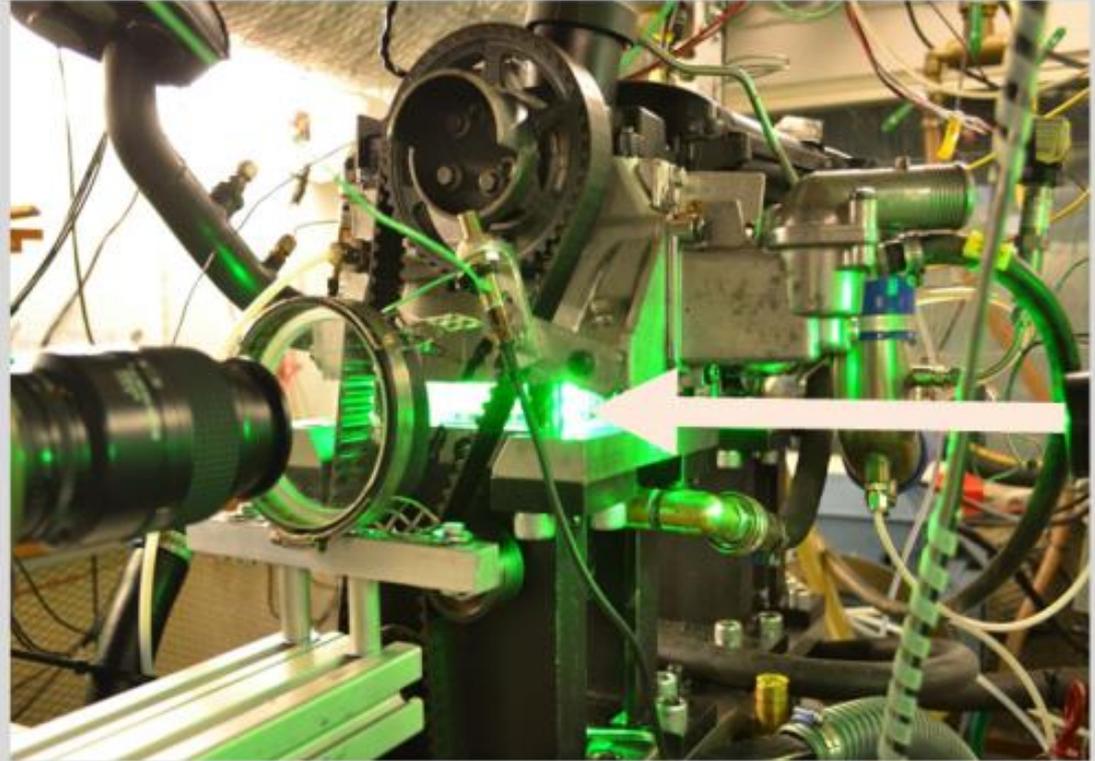
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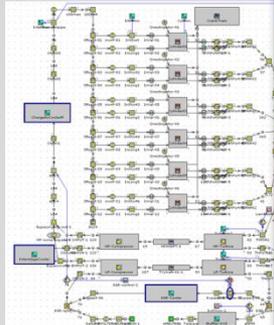


Investiga
efficiency

Optical diagnostics



System analysis



Investigate/explain overall V
Wheel PPC engine performan
methanol

Investigate/explain various
phenomenon of methanol
combustion inside the PPC engine

www.lth.se/mot2030

MOT-2030

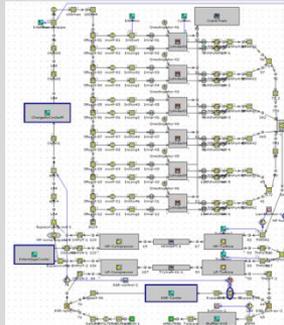
Highly Efficient Methanol Engine Systems
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Engine ex



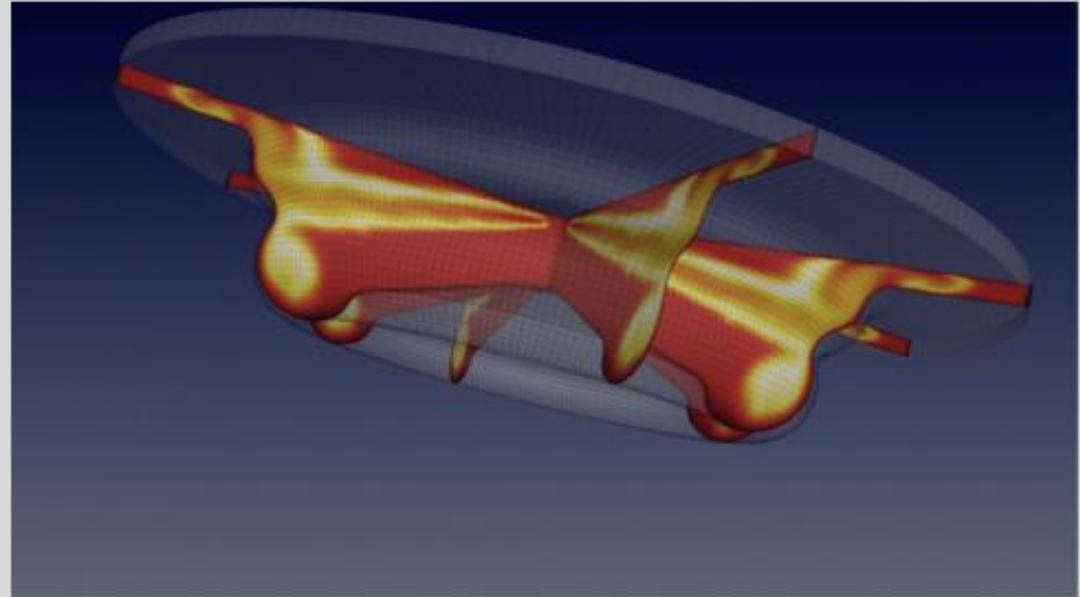
Investigate
efficiency p
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System analysis



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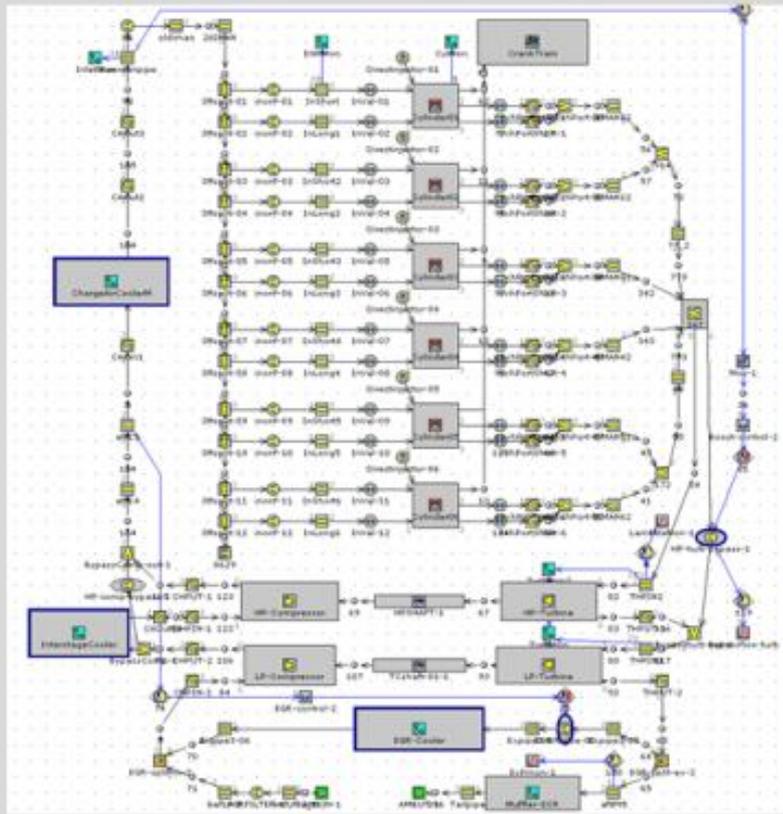
Advanced modeling



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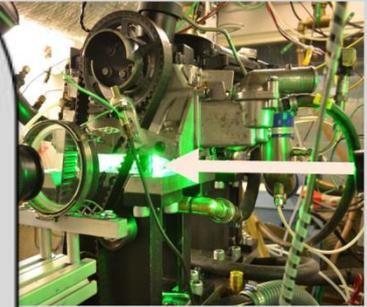
www.lth.se/mot2030

System analysis



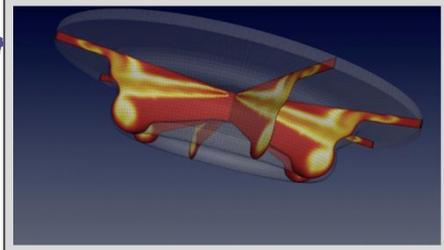
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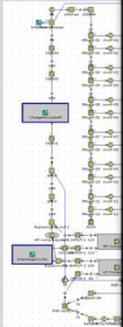
MOT-2030

Highly Efficient Methanol Engine Systems
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Supporting companies



System



Investigate/ex
Wheel PPC eng
n

Modeling



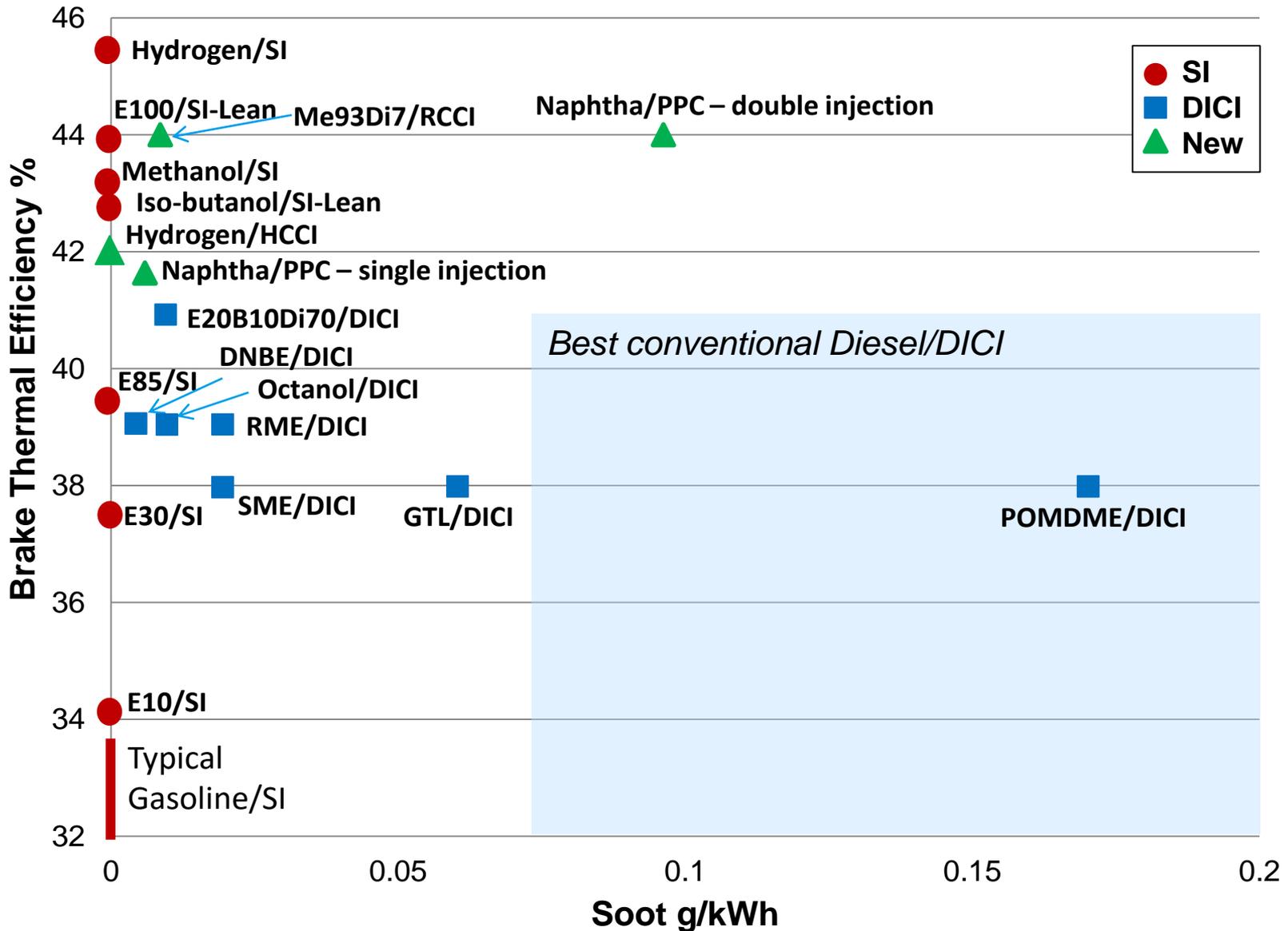
relations
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In-house tests and other experience
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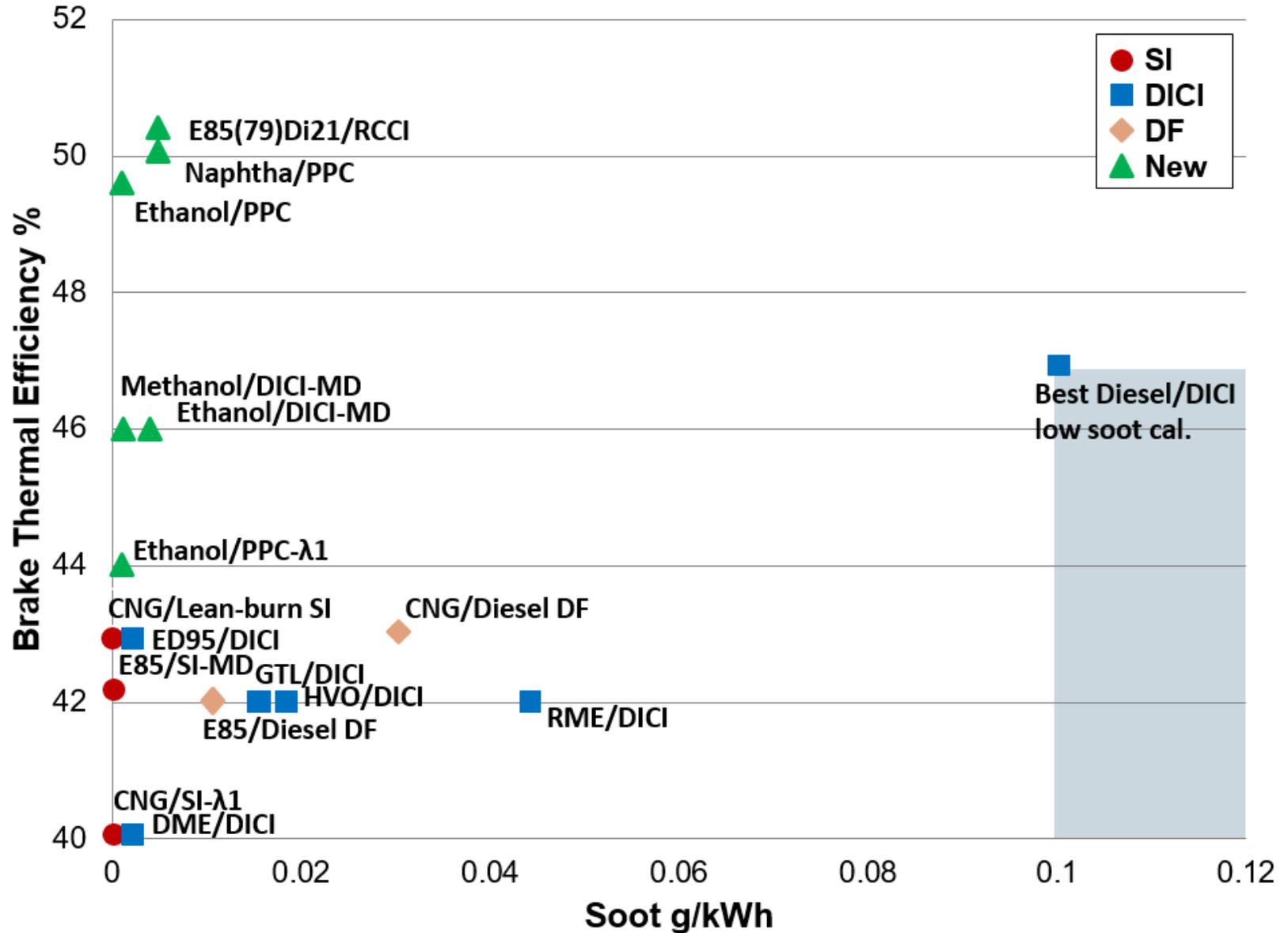
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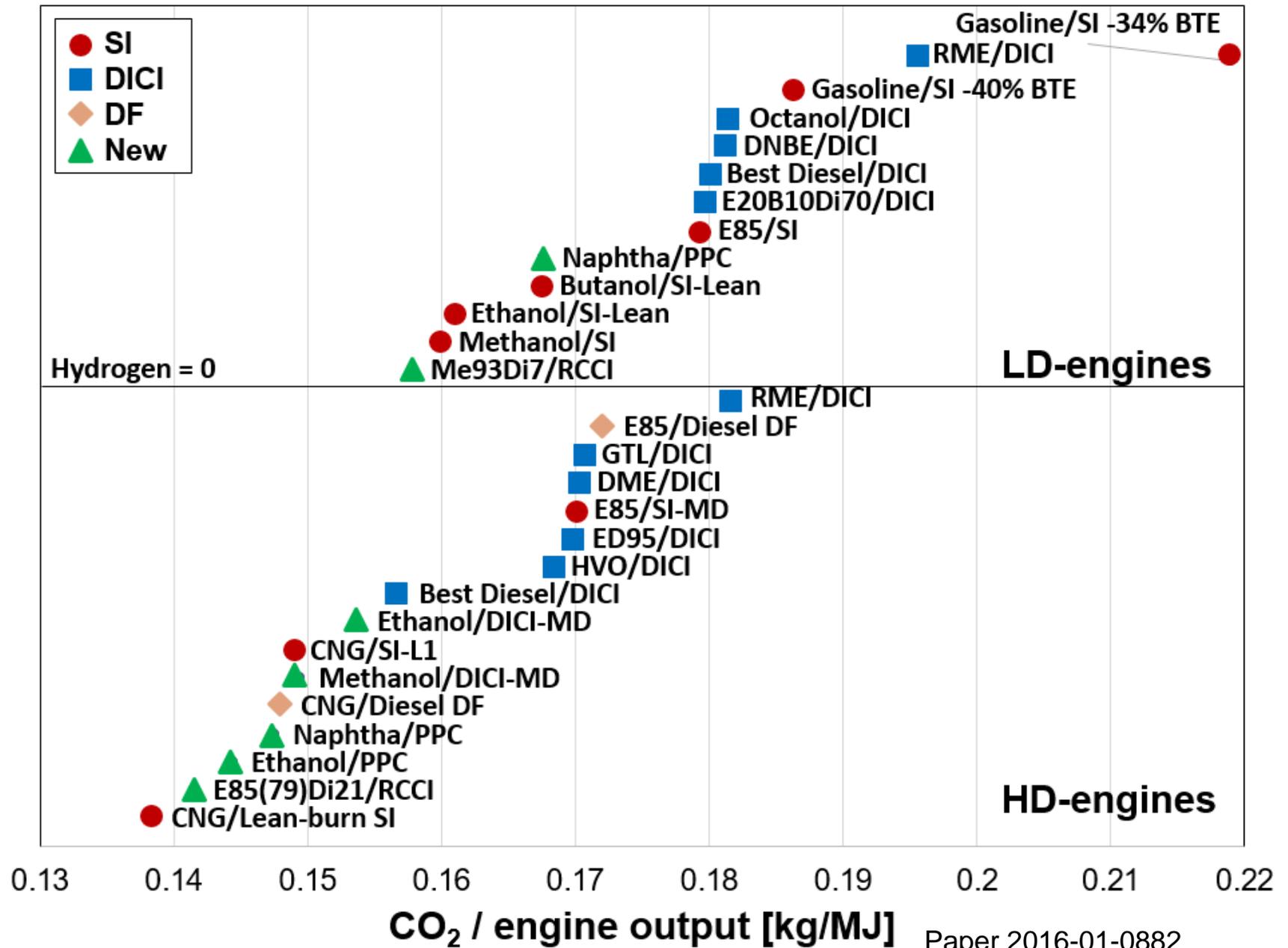
BTE and engine-out soot - LD engines



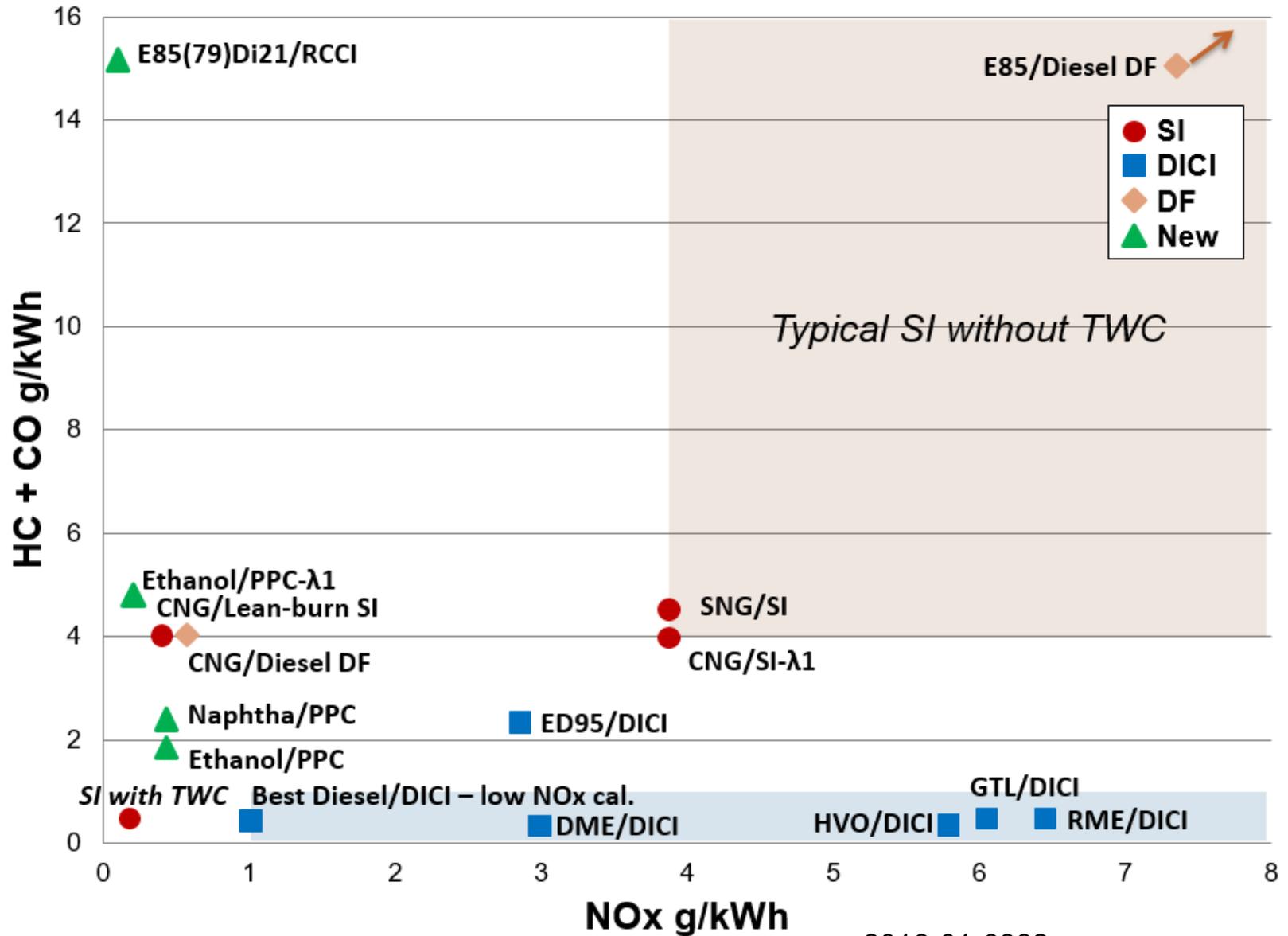
BTE and engine-out soot - HD engines



CO₂ tailpipe emissions, Tank-To-Engine output shaft



Engine-out emissions - HD engines



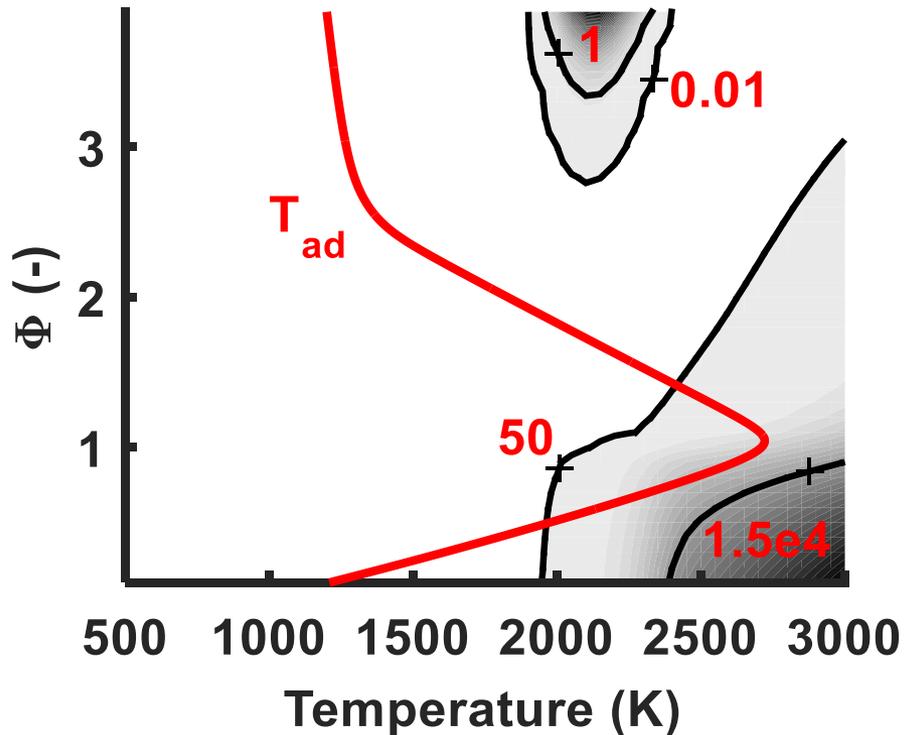
Theoretical Emissions Potential for Methanol Combustion



Theoretical NOx and Soot emissions

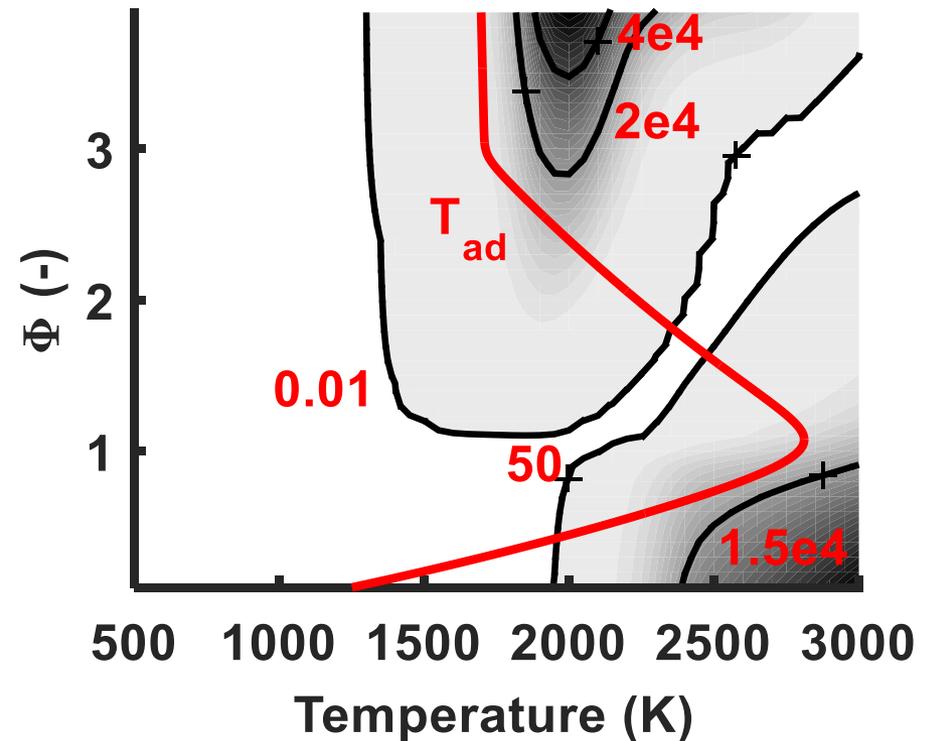
Methanol

$\Delta t = 9.00$ ms, $P = 60$ bar



Diesel

$\Delta t = 9.00$ ms, $P = 60$ bar



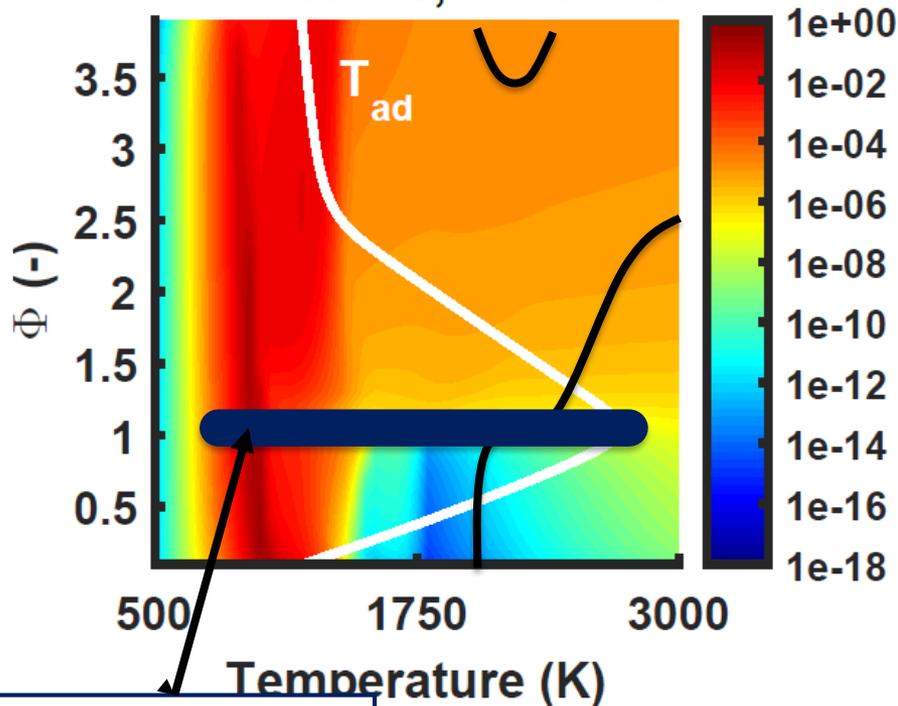
Svensson et al. Potential Emission Levels of Soot, NOx, HC and CO in the T-Phi Plane for Methanol, SAE 2016-01-0887



Why methanol engines have been associated with high formaldehyde emissions and formation of formic acid

Conventional MeOH premixed SI combustion

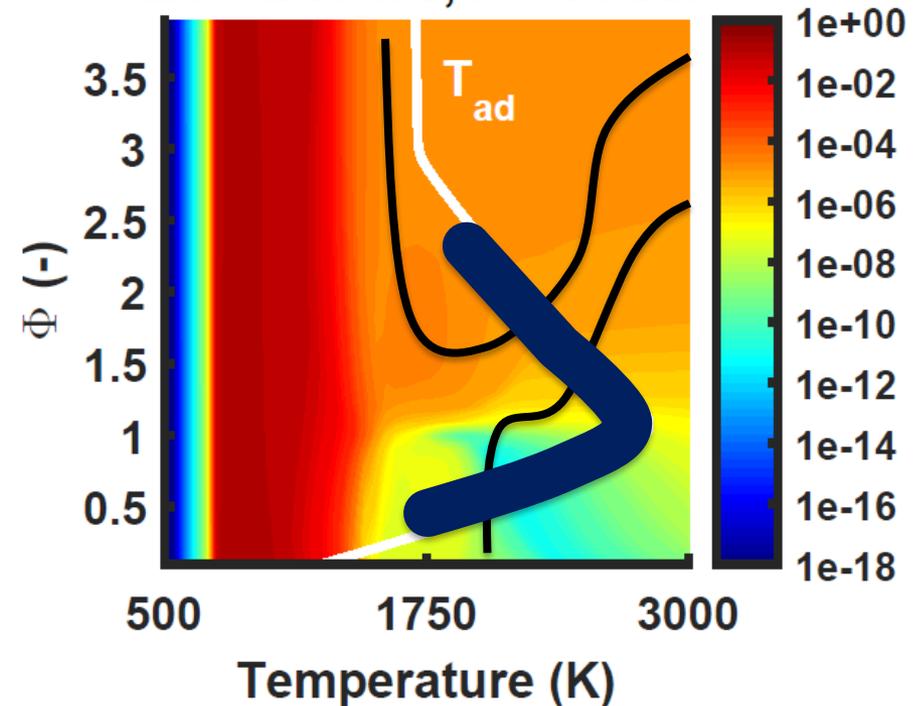
$\Delta t = 2.00$ ms, $P = 60$ bar



H_2O - Methanol

Conventional Diesel combustion

$\Delta t = 2.00$ ms, $P = 60$ bar

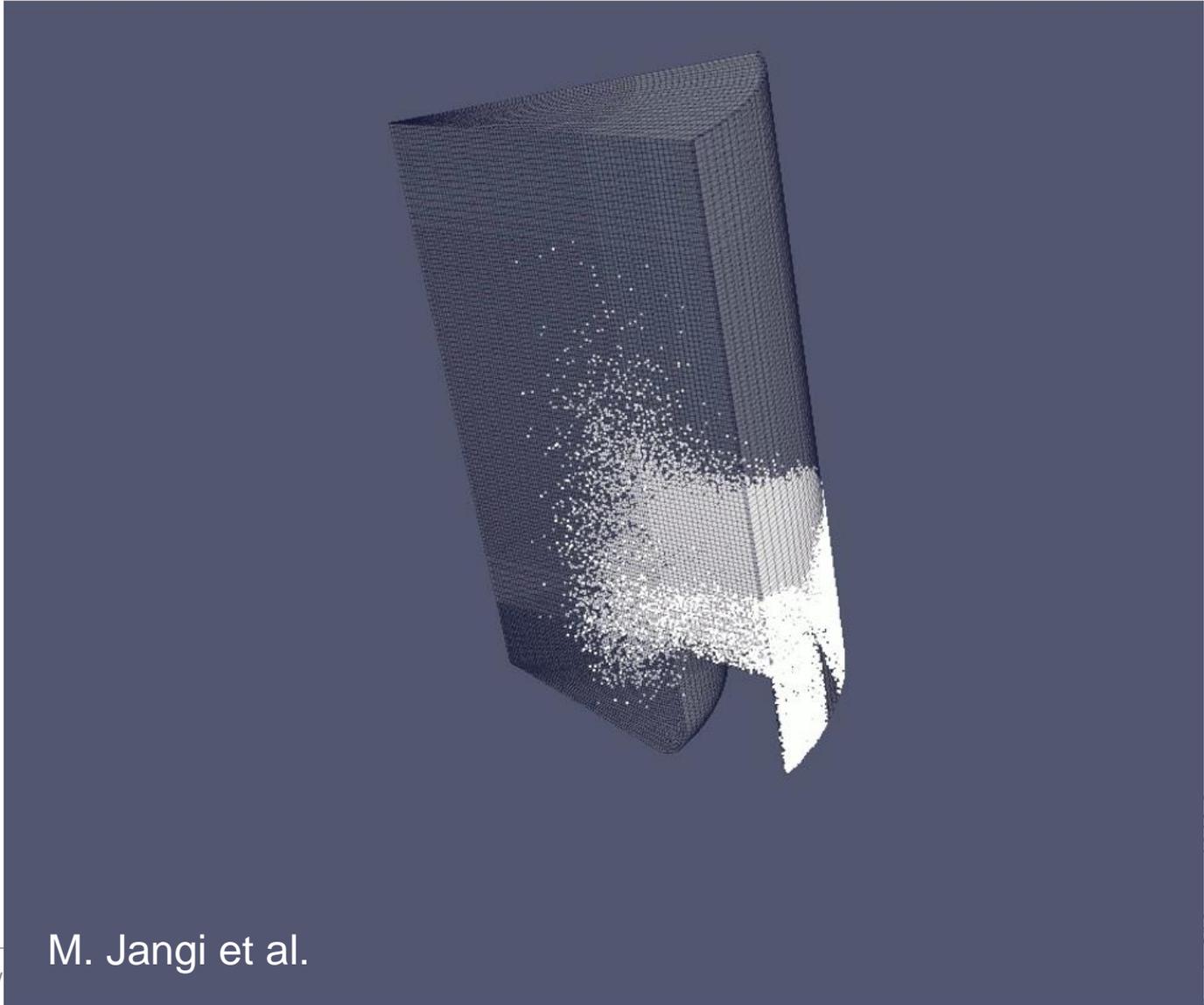


b) CH_2O - Diesel

Wall
quenching



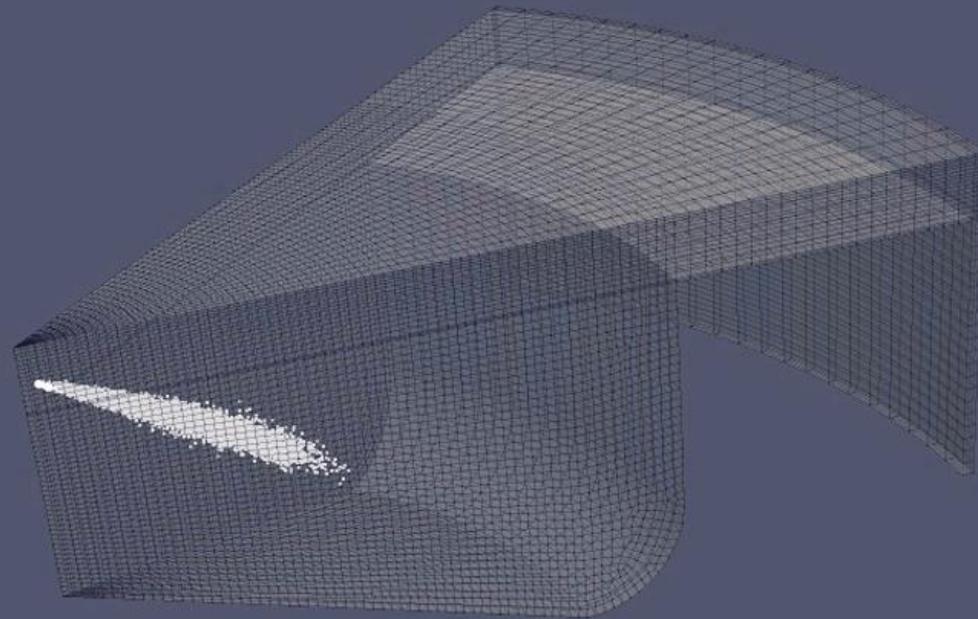
Early injection – premixed SI



M. Jangi et al.



Late injection – PPC



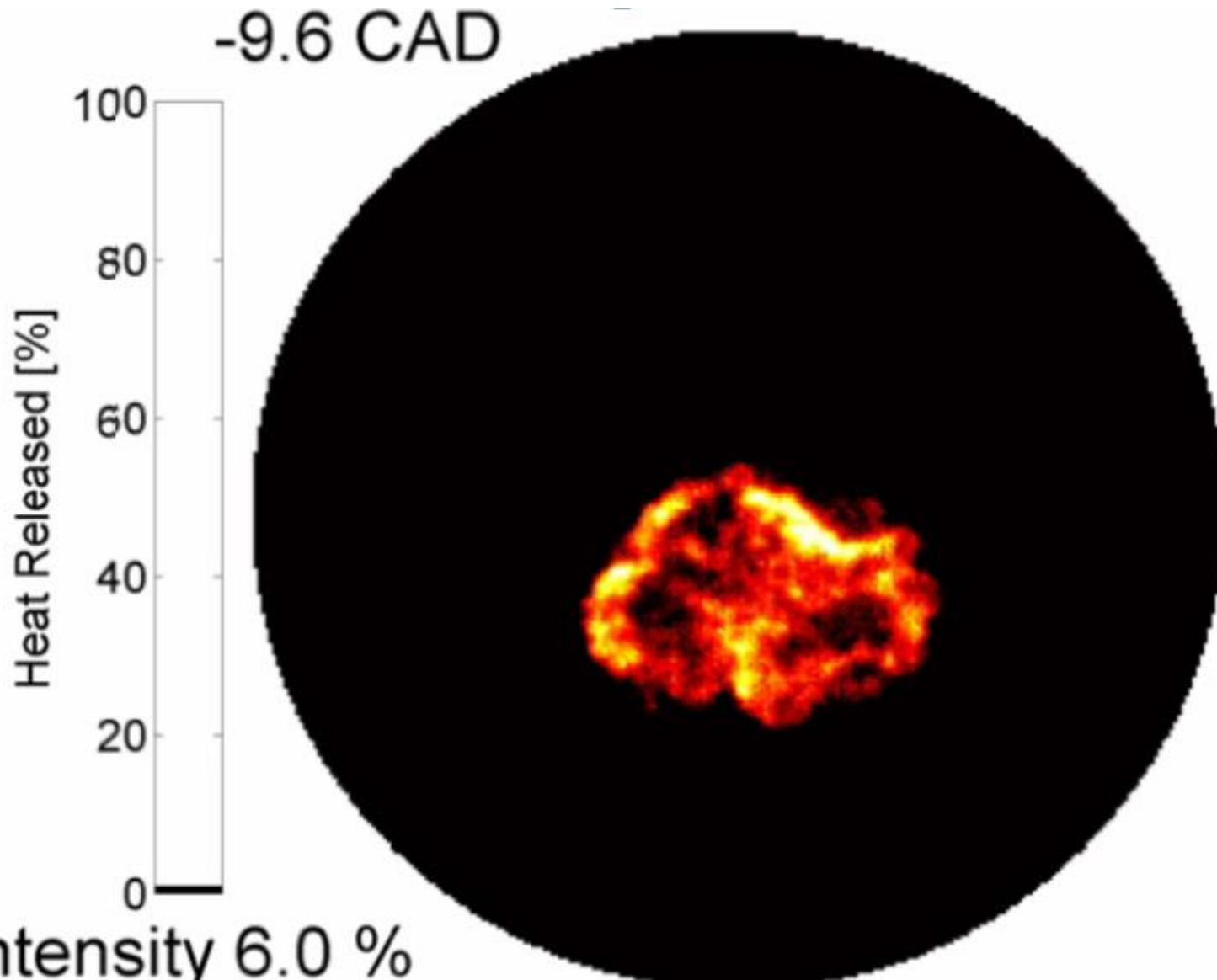
M. Jangi et al.



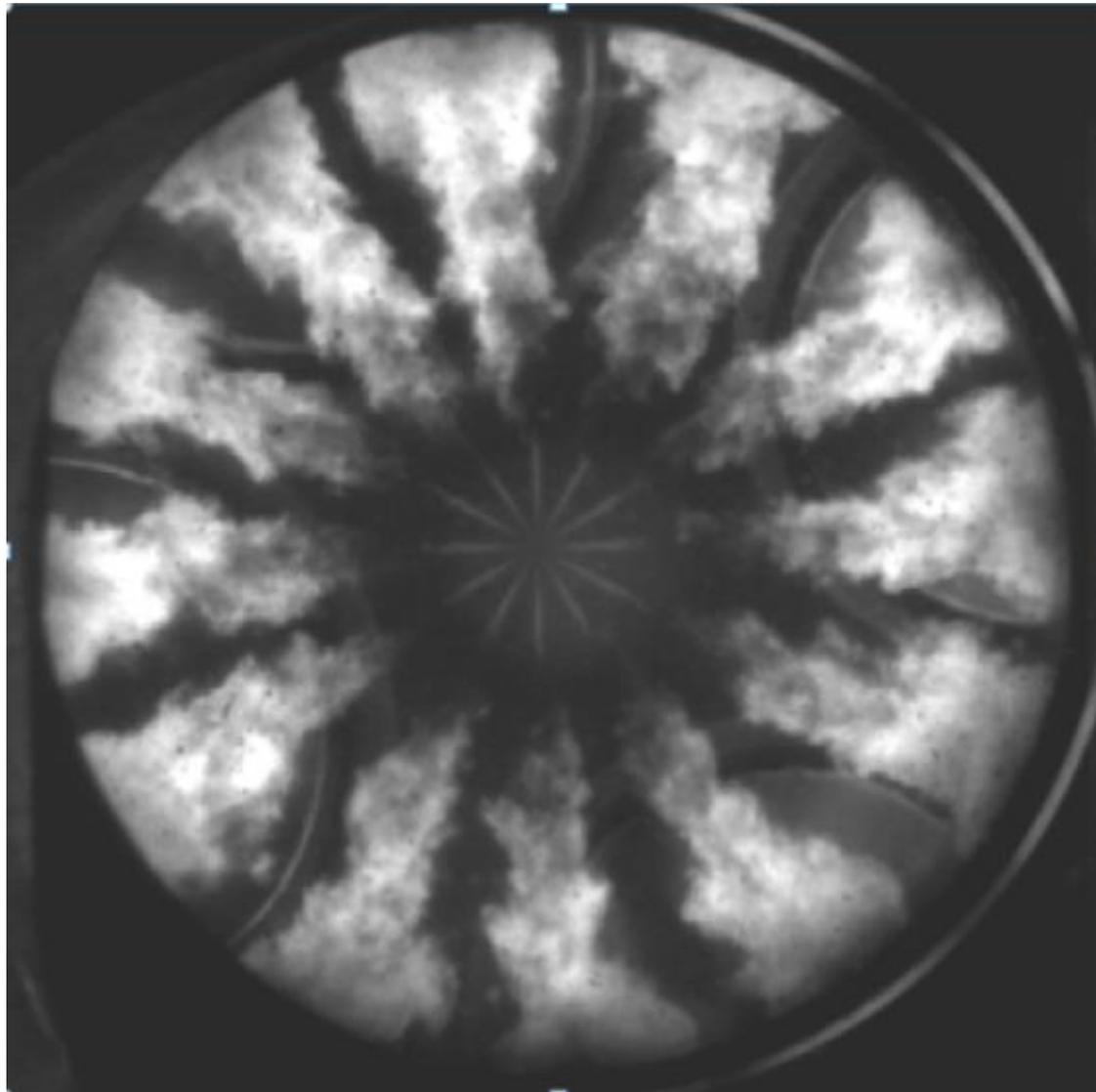
Methanol Engine Combustion Concepts Under Investigation

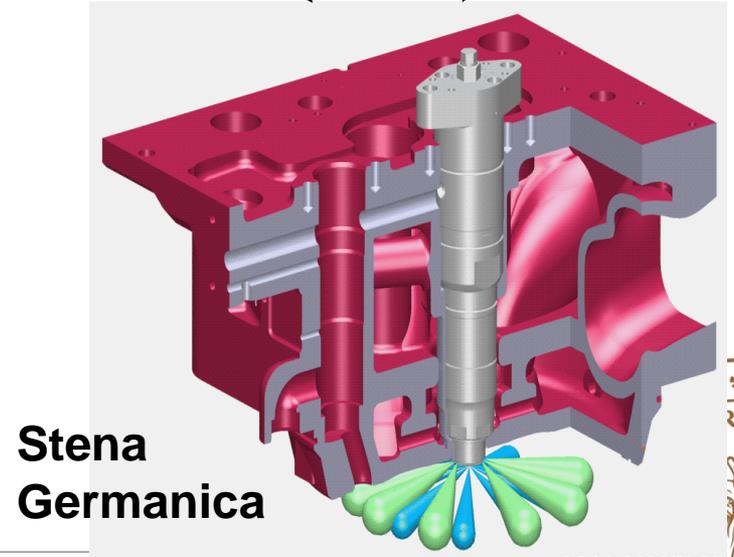
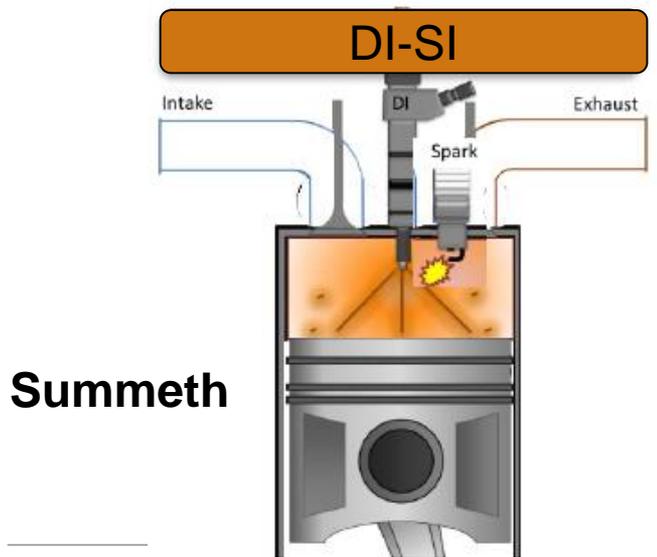
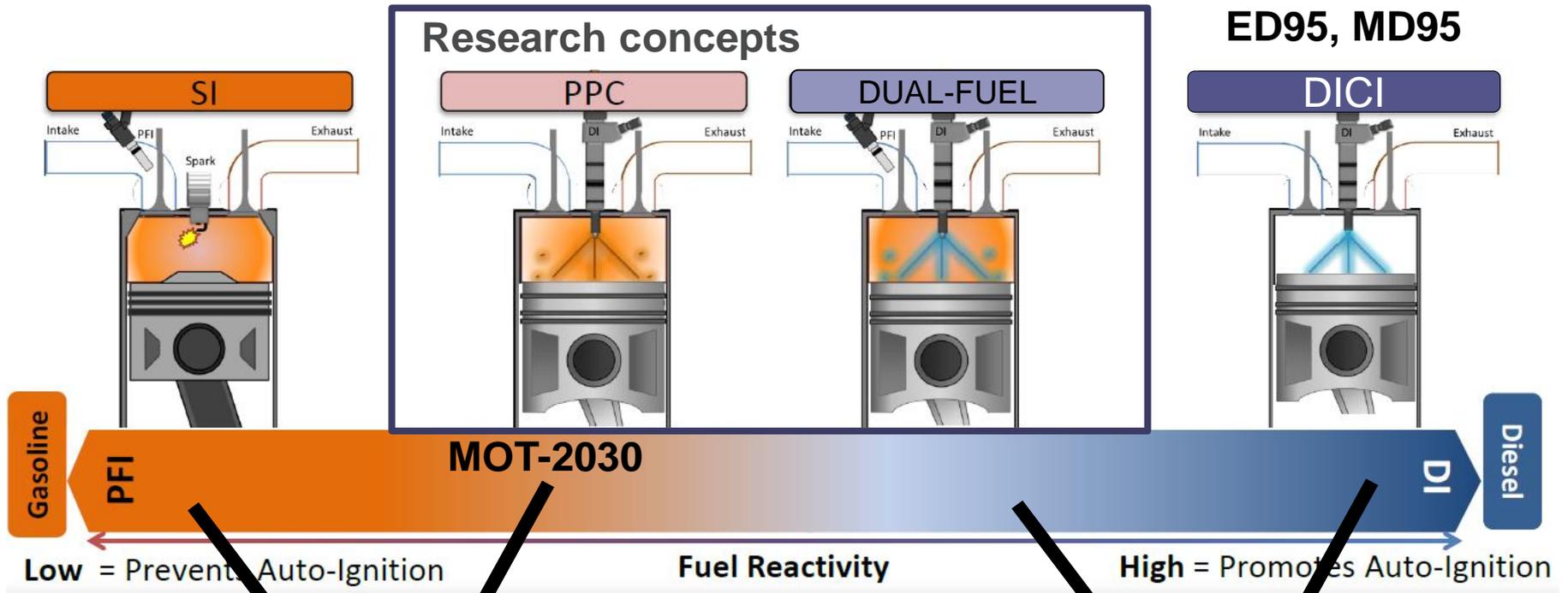


SI combustion – flame propagation



DI combustion – diffusion flame





Conclusions

Methanol

- One of the most promising candidates for low cost, high efficiency and low emissions operation
- Neat methanol combustion is soot-free

Direct Injection and Compression Ignition MeOH op.

- Dual fuel with direct injected diesel and port injected alcohols is established but lead to high emissions
- Wärtsiläs concept with double late direct injected dual fuel is functional and associated with reduced emissions

PPC

- High efficiency, low emissions and high load op. Fuel flexible.
- Still some immaturities as research concept: Low load - cold start issues, high combustion noise
- PPC-MeOH is possible



Observations & Research opportunities

- Late direct injection reduces formaldehyde and formic acid emissions – Something to pursue further in SI, DICI and PPC
- SI-MeOH will work. How well is the question!
- Is 60% gross efficiency within reach for HD-MeOH engines?
- Methanol has high sensitivity and high heat of vaporization – Is this combination a key feature in managing both high and low loads in compression ignition of neat methanol (for instance PPC)?

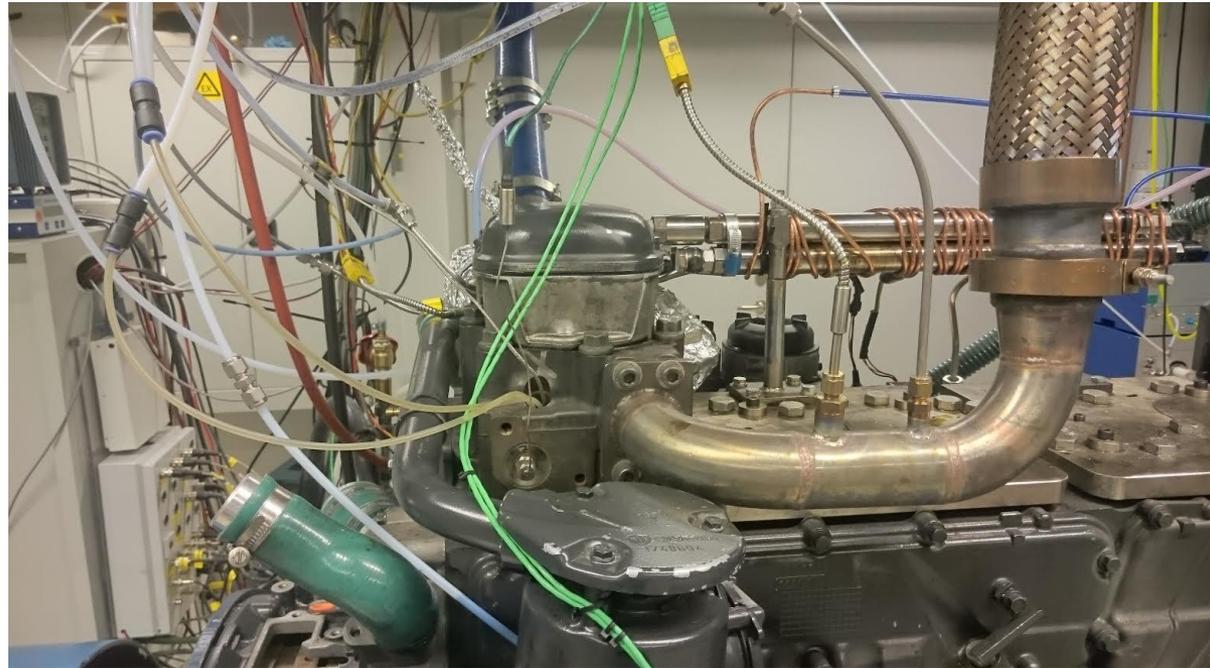


Extra slides

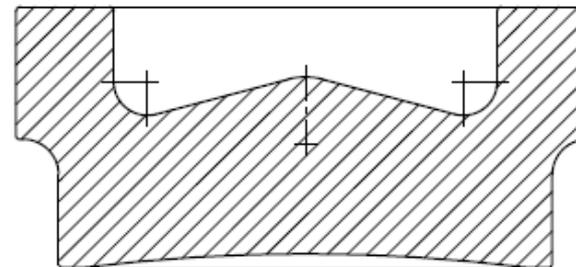


Test Rig

- Single cylinder Scania D13 Engine
- XPI injection system
- 10 hole injector with bigger holes for higher flow rate, 148° umbrella angle



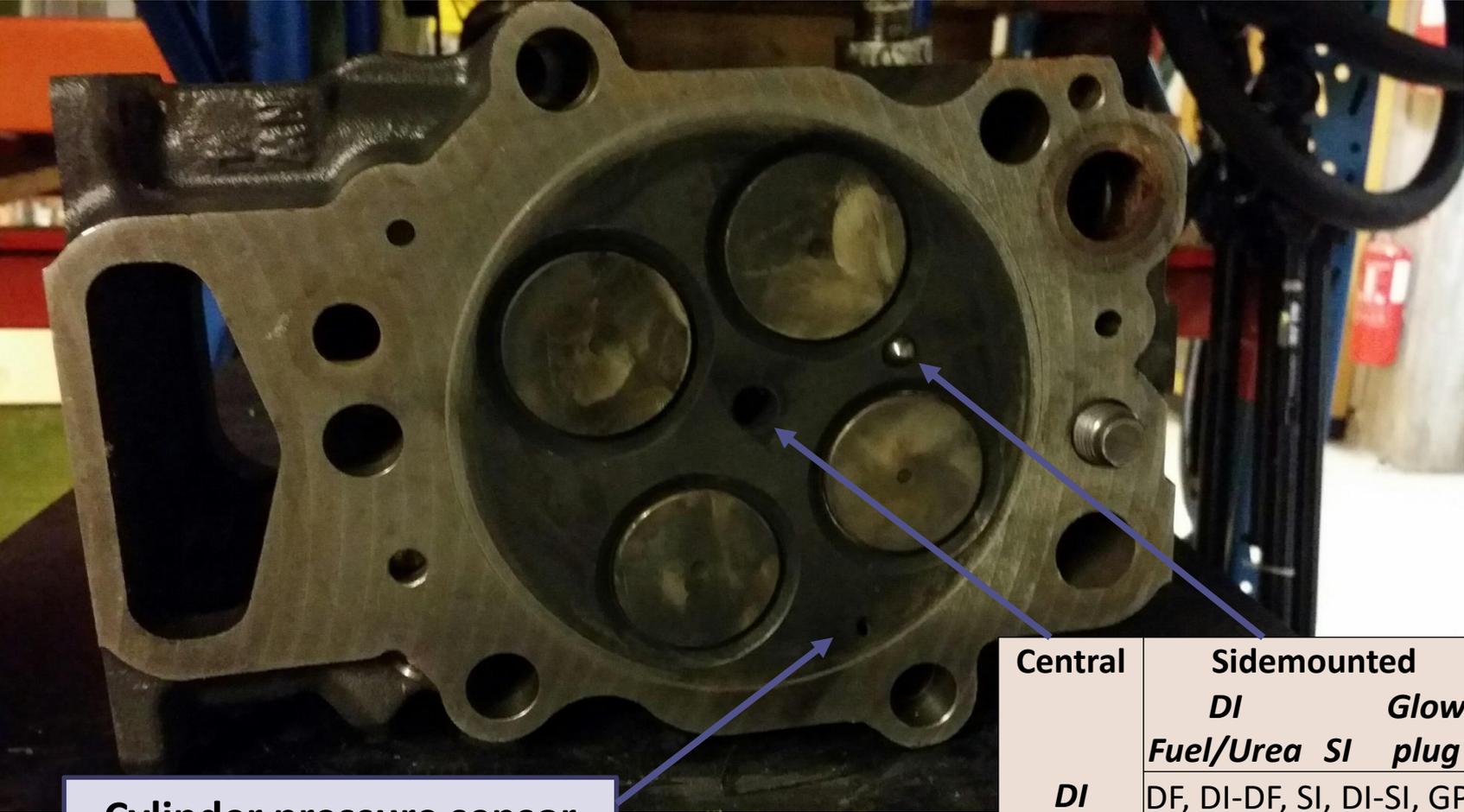
V_d	2124 cm ²
Stroke	160 mm
Bore	130 mm
r_c	15:1 nominal
Swirl ratio	2.1
IVC	-141 CAD ATDC
EVO	137 CAD ATDC



Piston Geometry



Multi-mode cylinder head



Cylinder pressure sensor

Central	Sidemounted	Port
	<i>DI</i>	<i>Glow plug</i>
	<i>Fuel/Urea SI</i>	<i>PFI</i>
<i>DI</i>	DF, DI-DF, SI, DI-SI, GP ass DI, DI-Urea etc etc	
<i>SI</i>		



What is PPC?

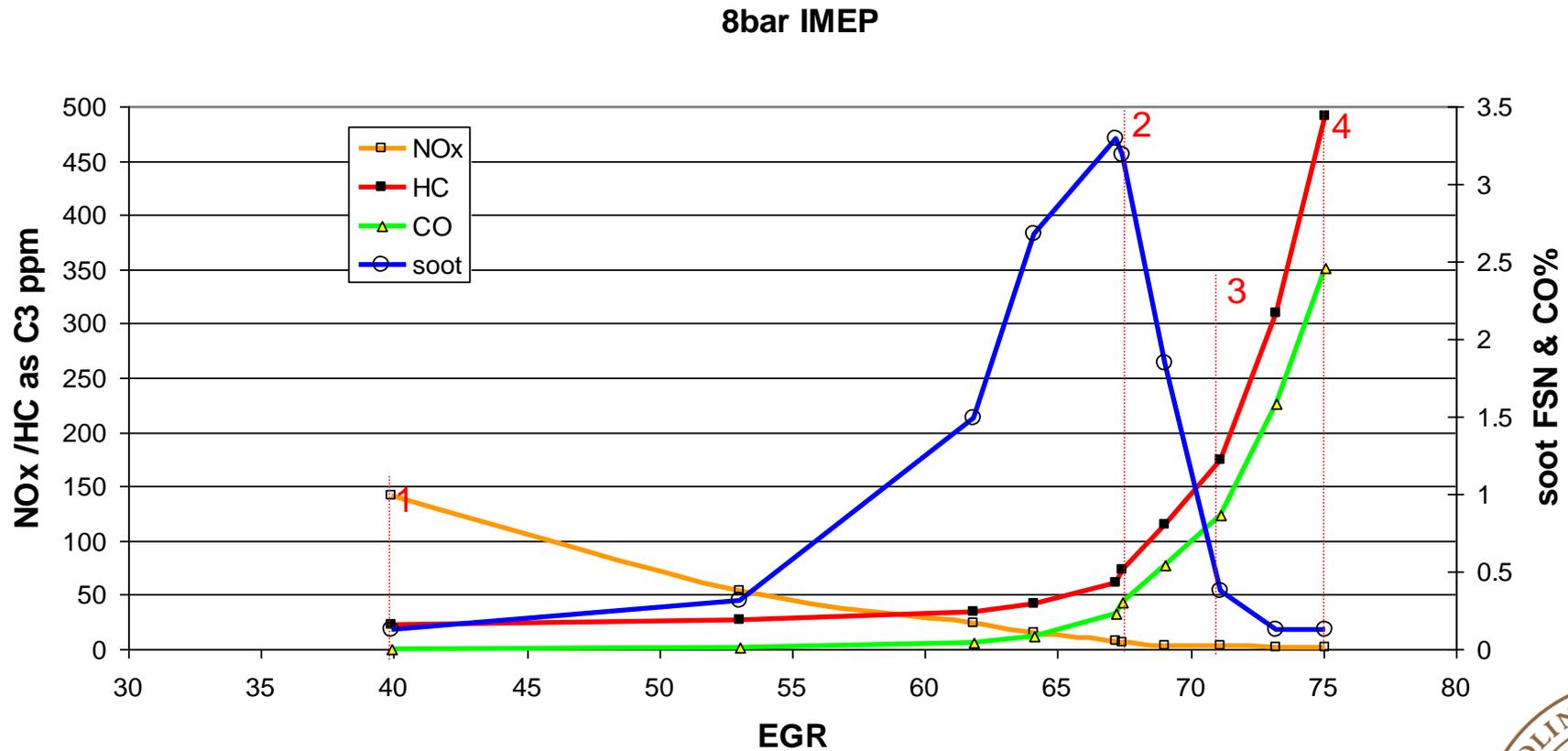
Partially Premixed Combustion

"The best fuel for a diesel engine is gasoline!"

Is methanol an even better "diesel" engine fuel?



Clue 1: EGR-sweep (diesel)

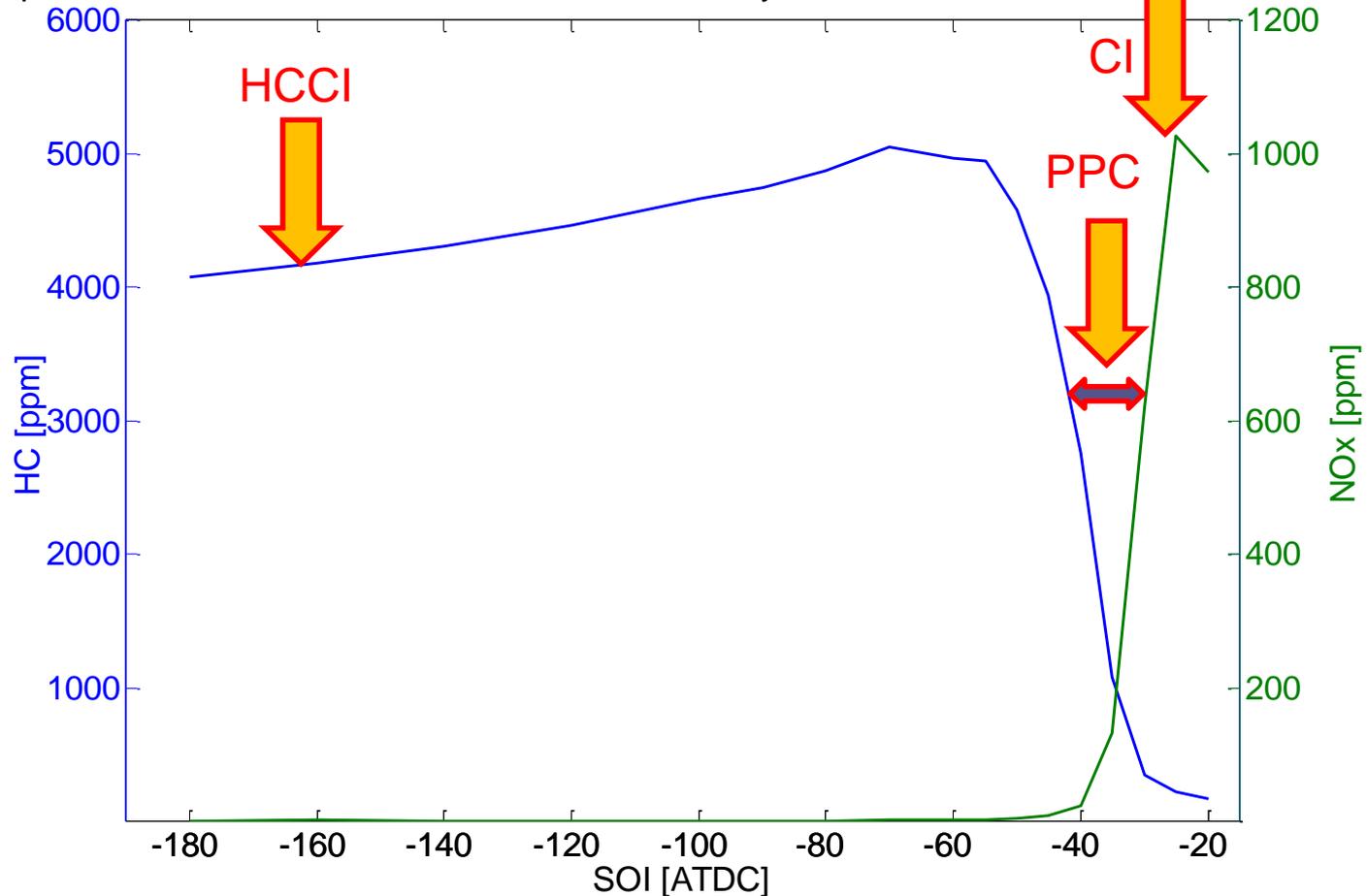


SAE 2006-01-3412



Clue 2: SOI sweep (DI)

Spridare 8x0.12x90 & 8x0.12x150, Iso-oktan, CR-tryck 750 bar, Duration 0,6 ms = 3.6 CAD



Def: region between truly homogeneous combustion, HCCI, and diffusion controlled combustion, diesel

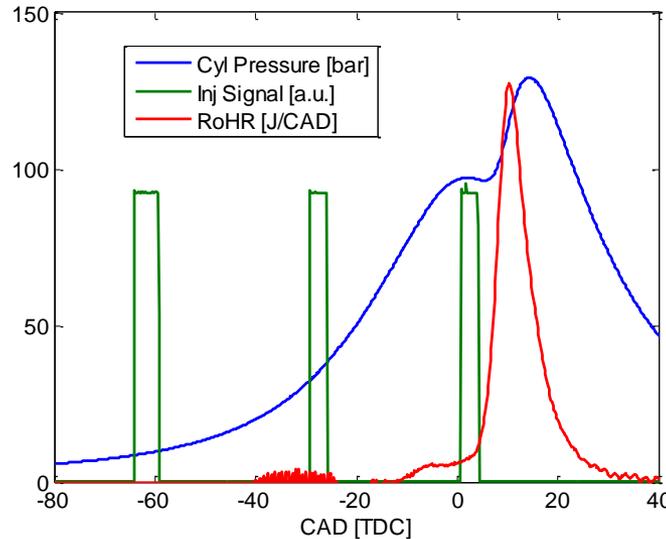


Clue 3: Burn rate and η_T

Optimum Thermodynamic efficiency

Low effective expansion ratio

High heat losses

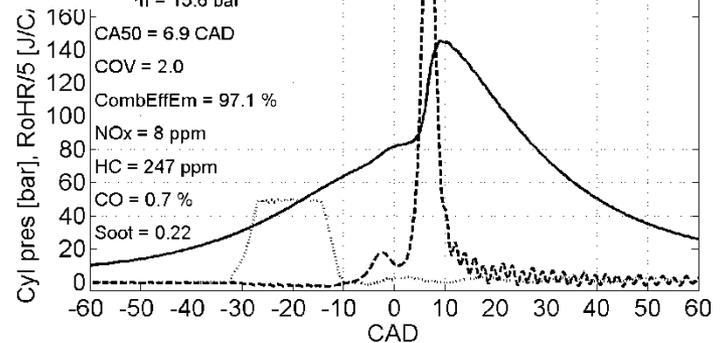


$\phi \times 0.18 \times 120$, SOI -33 ATDC, 1200 rpm, CR-pres 1500 bar

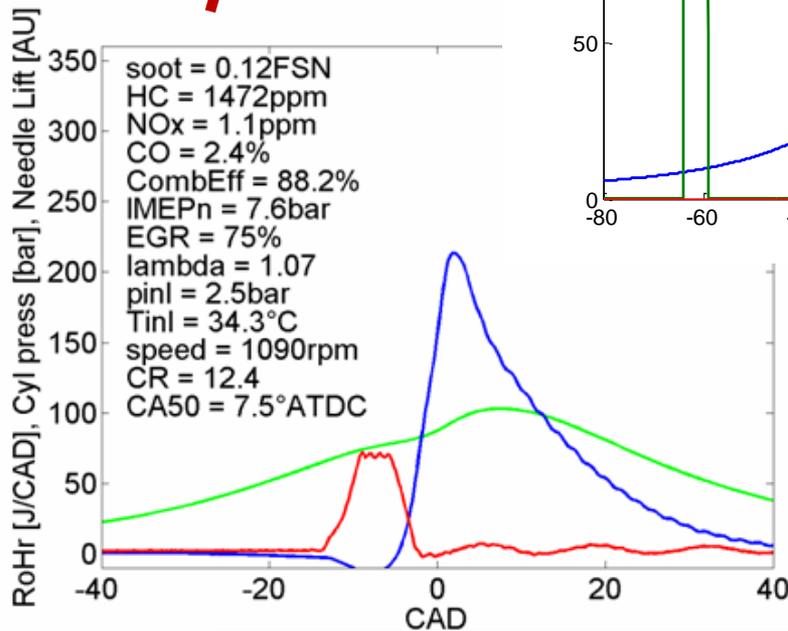
— Cyl pres [bar]
 - - - RoHR/5 [J/CAD]
 ... NeedleLift [AU]

Pinl = 2.42 bar
 Pexh = 2.46 bar
 Tintl = 22 °C
 Texh = 377 °C

$\eta_n = 15.6$ bar



CA50 = 6.9 CAD
 COV = 2.0
 CombEffEm = 97.1 %
 NOx = 8 ppm
 HC = 247 ppm
 CO = 0.7 %
 Soot = 0.22



soot = 0.12FSN
 HC = 1472ppm
 NOx = 1.1ppm
 CO = 2.4%
 CombEff = 88.2%
 IMEPn = 7.6bar
 EGR = 75%
 lambda = 1.07
 pinl = 2.5bar
 Tintl = 34.3°C
 speed = 1090rpm
 CR = 12.4
 CA50 = 7.5°ATDC

Premixedness



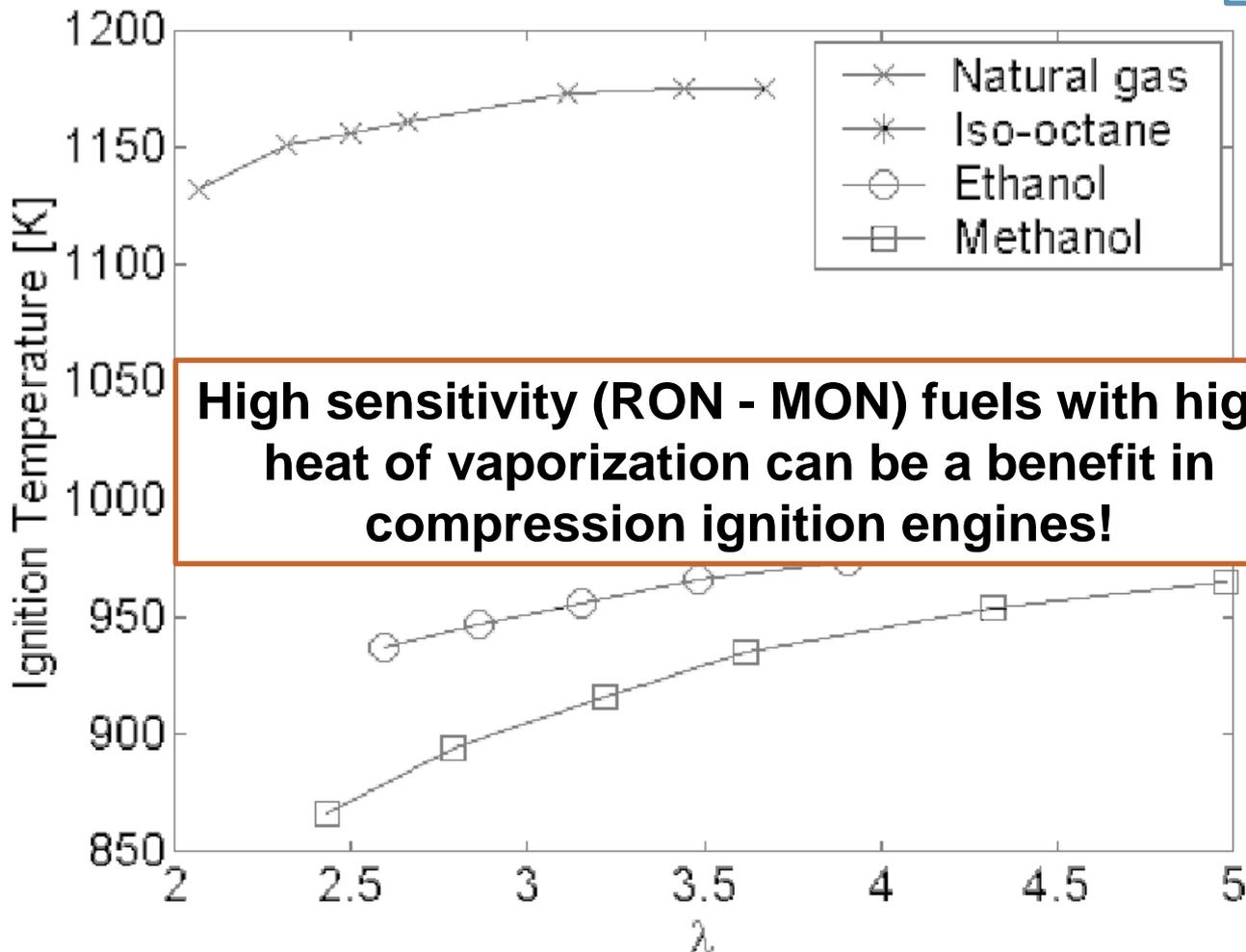
Lund PPC recipe

- ✓ $\lambda \approx 1.5$
- ✓ EGR $\approx 50\%$
- ✓ RON > 60
- ✓ SOI to control CA50



Ignition T of some fuels (very lean HCCI)

Premium Gasoline US
RON 92, MON 83



Natural gas
RON 120, MON 120

Iso-octane
RON 100, MON 100

Ethanol
RON 109, MON 90

Methanol
RON 109, MON 89

